

#### **1-Day Seminar on Pressure Vessel Integrity and Fatigue Failures**

SIRRIS, Technologiepark 935, 9052 Zwijnaarde, Ghent, Belgium <u>www.sirris.be</u> (hosted by SIRRIS)

## **Introduction to fatigue analysis**

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# Overview of presentation

- Fatigue mechanism
- Fatigue curves for welded and unwelded components
- Influencing parameters
- Damage cumulation : Miner rule
- Conclusion



## §17 Fatigue - Mechanism

## Fatigue of materials

Damage accumulation resulting of oscillating loadings (applications of  $\Delta\sigma/\Delta\epsilon$ )

## Two types of fatigue behaviour

Fatigue associated to a great number of cycles (main object of Clause 17)

Most usual case

Stress less that the flow stress

Rupture without noticeable deformation

Plastic Fatigue associated to a low number of cycles (oligocyclic fatigue) Case of the metallic wire folded and unfolded up to rupture Acceptable only in well controlled zones where the deformation is selflimitative and does not cause a distortion of the component

### Fatigue mechanism

Mechanism with local character

Each load cycle creates a damage accumulation

- Leading at term to
  - Crack initiation

Then crack propagation

Then to leak or failure



Slipping bands appear, creating intrusions-extrusions at the surface of the component



- A micro-crack resulting from stress concentration is formed
- If the stress range associated to the load variation is sufficient (greater than the endurance limit) the micro-crack propagates up to provoke rupture

In traction the crack propagates first at  $45^{\circ}$  (plan of maximum shear) then it changes its direction to progress perpendicularly to the direction of the most important principal stress ( $\rightarrow$ direction associated to the force of maximum opening)



## §17 - Fatigue - Facies

#### Macroscopic level

Presence of two distinct zones :

A smooth zone which corresponds to the zone of fatigue cracking A zone with grains associated to the brutal rupture happening at the end of the damage process









**Rotative Bending** 

The ratio of the surfaces of smooth zones and zones with grains is an indication of the margin towards rupture for the applied loading

## Microscopic level

Presence fatigue scratches which represent crack progression at each load cycle



Fatigue scratch

## **§17 - Fatigue – Fatigue strength**

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#### • Characterization of fatigue strength Each load cycle is characterized by its associated stress range $\Delta \sigma$

For a given stress range there is a number of allowable cycles defined by the applicable fatigue curve (S-N curve)

Fatigue damage and break will appear only if  $\Delta \sigma$  is greater than the endurance limit

It is necessary to apply corrections to the values issued from the fatigue curve to take account of:

> $\sigma_m$  (because curve  $\leftrightarrow \sigma_m = 0$ ) Stress concentrations Scale factor Surface state Temperature





#### Fatigue curve

## 🔅 🔰 §17 - Fatigue – Fatigue curve

- Welded Zones - 1

Curves are issued from works of IIW (International Institute of Welding

- They come from tests performed on representative welded assemblies
- Each welded assembly is associated to a fatigue curve named « fatigue class ». A fatigue class is identified by a number. This number is the value of the stress range that the welded assembly can bear two millions of times.

The curves take into account the influence of:
Local stress concentrations due to the geometry of the weld bead
Weld defects acceptable in manufacturing standards
Strong residual welding stresses
Non Destructive Testing (NDT) Procedures

The stresses to be used to exploit these curves are the linearised stresses at the foot of the weld bead

In practice, the Tresca equivalent stress computed at the foot of the weld bead (on the skin of the wall) is used. The equivalent stress is determined from the components of the linearised tensor of the stresses.





Curves for pressure vessels versus curves IIW

Curves of beads forbidden for pressure vessels were withdrawn Fatigue curves have a probabilistic nature. The curve of the test corresponds to a probability of failure of 50%. The curves of the Codes correspond to a lower probability of failure. The curves for pressure vessels correspond to a lower probability of failure (0,14%) than the IIW curves which have been established for civil engineering IIW curves have been established for priviled directions ( $\perp$  or // to the bead). In pressure vessels the lowest class IIW is considered and used with the equivalent stress (no more notion of direction).

## §17 - Fatigue – Fatigue curve – Welded Zone - 3

Rupture mode of welded zones

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Resistance of welded zones is less than that of the base metal Presence of geometrical singularities (→ stress concentrations) Presence of defects (bead shape, ....)

Crack initiation happens at bead foot It may happen at the root of a bead laid down of small thickness (< 0,8×min{thicknesses of welded components})



Fatigue with large number of cycles

Time to crack initiation is generally small. Life duration mainly depends on the speed of propagation of cracks. This propagation speed is not very dependent on the material ( $\rightarrow$  limited influence of R<sub>m</sub> of the material)

Oligocyclic Fatigue (NB : in principle curves do not apply directly !) Stress concentrations and residual stresses have a reduced effect and the material resistance (R<sub>m</sub>) has influence on the result

## §17 - Fatigue – Fatigue Curve –

## Welded Zone - 4

Influencing parameters to take into account

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Residual welding stresses (raw welding state) and mean stress

The presence of important residual stresses (near  $R_p$ ) makes that the mean stress has a limited influence on the result (this influence is difficult to quantify  $\rightarrow$  no correction de  $\sigma_m$ )

The presence of a corrective factor acting on the residual stresses (release, used welding process) is not taken into account

Stress Concentration : effect included in the fatigue curve



Same linearised stress for the same load. The different stress concentration is translated into a change of class



Ground weld: class 100 sustains  $\Delta \sigma$ =100MPa 2×10<sup>6</sup> times Not ground weld: class 80 sustains  $\Delta \sigma$ =80MPa 2×10<sup>6</sup> times

#### Material strength

Limited influence in fatigue with great number of cycles ( $\rightarrow R_m$  has no influence : the use of a high resistance steel does not modify the result)

Scale factor: tests are performed on assemblies of thickness 25 mm ( $\rightarrow$  the scale correction contains terms in (25/e)<sup>n</sup>)

## **PFRC** 💭 §17 - Fatigue - Fatigue Curve –

Welded Zone- 5

European Pressure Equipment Research Council Inluencing parameters to take into account (continuation)

l'emperature

#### Surface condition

For some beads the class used is a function of the presence or not of a finishing process (NB : No corrective factor in simplified fatigue analysis)

Influence of weld defects and controls

For some beads the class used is function of the presence or not of a control (class is different if the zone can be controlled or not)

Fatigue curves come from tests performed on assemblies controlled to be exempt of major defects. Any defect has a crucial influence on the fatigue behaviour: a shape defect leads to a stress concentration, the other defects can be assembliated to pre-existing cracks

Thus the presence of a control constitues one preliminary step necessary to any correct exploitation of the fatigue curves



Crack in foot caused by a shape defect Crack in root caused by a lack of penetration



## **D ()** §17 - Fatigue – Fatigue Curve –

## **Base Metal - 1**

European Pressure Equipment Research Council Curves are inspired from those of AD-Merkblatt S2

They are associated to a probability of failure de 0,1%. In comparison to curves associated to a probability of failure of 50% they incorporate:

A safety factor of 10 on the number of admissible cycles

A safety factor of 1,3 on the stress range

The curves of AD use a safety factor of 1,5 on  $\Delta \sigma$ . This factor has been reduced because: With 1,5 several curves were under the best curve of welded assemblies (Class 100) EN 13445-3 recommends an inspection at the end of a period less than 20% of the allowable fatigue life (cf. EN 13445-3 Annexe M) while AD uses a period of 50% of life duration

The stresses which must be used to exploit those curves are the total « effective » stresses (total  $\rightarrow k_t \rightarrow k_f \rightarrow$  effective) in the zone under consideration

In practice the designer uses the equivalent stress of Tresca computed from the components of the stress tensor

Calculation of total stresses : 2 possibilities

Use of linearised stresss multiplied by a factor of stress concentration  $(k_t) \leftrightarrow$  use of a model which does not incorporate the exact geometry of the weld bead Use of total stresses  $\leftrightarrow$  use of a model which incorporates the exact geometry of the weld bead

## **FRC** §17 - Fatigue – Fatigue curve –

## **Base Metal - 2**

European Pressure Equipment Research Council The endurance limit corresponds to  $2 \times 10^6$  cycles and cut-off to  $10^8$  cycles

- The curves show a fatigue life duration dominated by the time necessary to initiate a crack  $(\rightarrow \text{ response fonction of the resistance of the material})$
- Influent Parameters to take into account

#### Mean stress

Life duration is shortened in presence of a mean stress of traction

If increases in presence of a mean stress of compression

The correction takes account of this phenomenon, it is that of AD-Merkblatt S2

Stress concentration: in case of use of linearised stresses

#### Resistance of material

The net of curves is parametered in fonction of the value of R<sub>m</sub>

Scale factor : tests are performed on specimens of thickness 25 mm ( $\rightarrow$  scale correction contains terms in (25/e)<sup>n</sup>)

Temperature

#### Surface

The correction is that of AD-Merkblatt S2

## **EPERC** §17 - Fatigue - Courbe de fatigue – Métal de base - 3

European Pressure Equipment Research Council Influent parameters to take into account (suite)

#### Pasticity correction

In the domain of oligocyclic fatigue, fatigue curves are established in fonction of the imposed strain variation  $\Delta \epsilon$ 

As long as  $\Delta \sigma \leq 2 \times R_{pt} \Delta \sigma$  et  $\Delta \epsilon$  are proportionnal after adaptation and the  $\Delta \epsilon$  obtained from a calculation in linear elasticity in writing  $\Delta \sigma = E \times \Delta \epsilon$  is correct, it can be used directly to exploit the fatigue curves

If  $\Delta \sigma > 2 \times R_{pt}$  the  $\Delta \epsilon$  computed from a calculation performed in linear elasticity is under estimated (while  $\Delta \sigma$  is over estimated).  $\Delta \epsilon$  must be corrected using a amplifying coefficient noted  $k_e$  ( $k_v$  for the thermal part- because the evolution profile of these stresses in the thickness is different -  $k_v < k_e$ )

By definition  $k_e = \Delta \varepsilon_{ep} / \Delta \varepsilon_{elastique}$ ;  $\Delta \varepsilon_{ep}$  is the real elastoplastic deformation

The correction coefficient k<sub>e</sub> used is that of AD-Merkblatt S2

Particular case of simplified rules

Only one fatigue curve

Curve of P235GH steel (lowest  $R_m$ )

The curve incorporates the effect of mean stress ( $\rightarrow$  no correction of  $\sigma_m$ )

It is built for the largest mean stress acceptable:  $\sigma_m = R_{pt} - (\Delta \sigma/2)$ 

The curve incorpores the effect of surface state ( $\rightarrow$  no associated correction) It is built for the state of delivery of the material

## 💊 💭 §17 - Fatigue –

**Influencing Parameters** 

#### Stress concentration

Every stress concentration coming from the design of the component (notch, small radius of connexion, ...) reduces fatigue resistance

The ratio betweeen  $\sigma$  maximum acting at the level of the section where is the concentration and the  $\sigma$ which would act uniformly at the level of the same section is called factor of concentration of stress (K<sub>i</sub>)

#### $(K_t > 1)$

The notch factor  $(K_f)$  is for a given life duration the ratio between acceptable  $\Delta \sigma$  of the component without notch and acceptable  $\Delta \sigma$  of the component with notch

 $\Delta \sigma_{\text{notch}} = \Delta \sigma_{\text{without-notch}} / K_f$ K<sub>f</sub> > 1 et K<sub>f</sub> = fonction(K<sub>t</sub>) in general K<sub>f</sub> < K<sub>t</sub>

#### Scale factor

Endurance limit is reduced when the size of the component is increased. A component of large size contains potentially more defects. This increases the number of sites of crack initiation.



#### **EPERC European Pressure Equipment Research Council Surface state Surface state} Surface state}**

Endurance limit depends on the state of surface of the component. It reduces when the surface state is degraded.

A component having a bad surface state has more sites of crack initiation

This is important since the maximum stresses appear in general at the level of the skin

#### Effect of temperature

For current steels a temperature increase provokes a reduction of the endurance limit

#### Influence of load history

A stress cycle of amplitude inferior to the endurance limit must be taken into account (it provokes a non-negligible damage) if it occurs after a stress cycle of a larger range than the endurance limit





# Conclusion

- The different fatigue mechanisms to be considered in the construction of pressure vessels have been presented
- It has been shown how fatigue influencing parameters are taken into account in a conservative manner by the simplified analysis of Clause 17 of EN 13445



## Thank you for your attention